

## ESZ type 150 | with national technical approval

### General information and calculation basis

#### Character meanings in the formula

$f_{tRd}$	=	Calculated value of the internal resistance of the bearing [N/mm <sup>2</sup> ]
$G$	=	Shear modulus [N/mm <sup>2</sup> ]
$a$	=	shorter side of the bearing [mm]
$b$	=	Longer side of the bearing [mm]
$t$	=	Bearing thickness [mm]
$\alpha_{bd}$	=	Angular rotation around the axis parallel to the bearing side b [‰]
$\alpha_{ad}$	=	Angular rotation around the axis parallel to the bearing side a [‰]
$c$	=	Mainly loaded bearing side of a rectangular bearing [mm]
$\eta_2$	=	Aspect ratio coefficient

$b/a$	$\eta_2$
1	0,208
1,5	0,231
2	0,246
3	0,267
4	0,282
6	0,299
8	0,307
10	0,313
$\infty$	0,333

Table 1:  
 $\eta_2$  as a function of the aspect ratio  $b/a$  as a table of values  
 (Intermediate values may be interpolated linearly)

$$f_{tRd} = \frac{R_{\perp d} \cdot t}{\eta_2 \cdot a}$$

$f_{tRd}$  is the calculated value of the internal resistance of the bearing and is used to calculate the permissible compressive stress  $\sigma_{z,Rd}$

	Shape factor range $S$ ( $S$ , $S_{\text{borehole}}$ or $S_{\text{mod}}$ )	Function for determining the design value of the load-bearing capacity $R_{\perp d}$ [N/mm <sup>2</sup> ]
Point and strip bearings	0.88 – 5.00	$R_{\perp d} = 7.22 \cdot S - 3.39$
	5.00 – 7.00	$R_{\perp d} = 8.95 \cdot S - 12.02$
	7.00 – 10.00	$R_{\perp d} = 1.96 \cdot S + 36.86$
	$\geq 10.00$	$R_{\perp d} = 56.50$

Table 2:

$R_{\perp d}$  = Rated value of the associated load-bearing capacity of the bearing [N/mm<sup>2</sup>] perpendicular to the bearing plane as a function of the shape factor  $S$  at a compression  $\epsilon = 40$  %.

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#### Initial assumptions

$F_{z,max,d}$	=	200	kN
$f_{tRd}$	=	Formula	N/mm <sup>2</sup>
<b>a</b>	=	130	mm
<b>b</b>	=	150	mm
<b>t</b>	=	15	mm
<b>G</b>	=	1.2	N/mm <sup>2</sup>
$\alpha_{Statics}$	=	5.2	‰
$\alpha_{Obliqueness}$	=	10	‰
$\alpha_{Unevenness}$	=	4.8	‰
$\alpha_{bd\ total}$	=	20	‰
$\eta_z$	=	0.215	

In this calculation example, a rotation around the axis parallel to the bearing side b ( $\alpha_{bd}$ ) is calculated. The bearing has no holes.

#### Calculation way

$$S = \frac{130 \cdot 150}{2 \cdot 15 \cdot (130 + 150)} = 2.32$$

$$R_{\perp d} = 7.22 \cdot S - 3.39 = 17.22 \cdot 2.32 - 3.39 = 13.36 \text{ N/mm}^2$$

$$f_{tRd} = \frac{13.36}{0.215} \cdot \frac{15}{130} = 7.18 \text{ N/mm}^2$$

$$\sigma_{z,Rd} = \left[ 7.18 - 0.02 \cdot \frac{1.2}{2} \cdot \left( \frac{130}{15} \right)^2 - 0 \cdot \frac{1.2}{2} \cdot \left( \frac{150}{15} \right)^2 \right] \cdot \frac{130}{15} \cdot 0.215 = 11.70 \text{ N/mm}^2$$

$$\sigma_{z,m} = \frac{200.000}{130 \cdot 150} = 10.26 \text{ N/mm}^2$$

$$\sigma_{z,Rd} = 11.70 \text{ N/mm}^2 \geq \sigma_m = 10.26 \text{ N/mm}^2 > \text{Proof provided!}$$