## ESZ type 100 | with national technical approval General information and calculation basis

## Character meanings in the formula

```
f
ftRd = resistance of the bearing [N/mm}\mp@subsup{}{}{2}
G = Shearmodulus[N/mm
a = shorter side of the bearing [mm]
b = Longer side of the bearing [mm]
t = Bearing thickness [mm]
\mp@subsup{\alpha}{bd}{}}=\mathrm{ Angular rotation around the axis
\mp@subsup{\alpha}{bd}{}}
ad}=\mathrm{ Angular rotation around the axis
\mp@subsup{\alpha}{ad}{}}=\quad\mathrm{ parallel to the bearing side a [%o]
c = Mainly loaded bearing side of a
c rectangular bearing [mm]
\eta
```

| b/a | $\eta_{2}$ |
| :---: | :---: |
| 1 | 0,208 |
| 1,5 | 0,231 |
| 2 | 0,246 |
| 3 | 0,267 |
| 4 | 0,282 |
| 6 | 0,299 |
| 8 | 0,307 |
| 10 | 0,313 |
| $\infty$ | 0,333 |

Table 1:
$\eta_{2}$ as a function of the aspect ratio b/a as a table of values
(Intermediate values may be interpolated linearly)
$f_{t R d}=\frac{R_{\perp d}}{\eta_{2}} \cdot \frac{t}{a} \quad \begin{aligned} & f_{\text {trd }} \text { is the calculated value of the internal resistance of the bearing and is used } \\ & \text { to calculate the permissible compressive stress } \sigma_{z, R d}\end{aligned}$

|  | Shape factor range $S$ ( $\mathrm{S}, \mathrm{S}_{\text {borehole }}$ Or $\mathrm{S}_{\text {mod }}$ ) | Function for determining the design value of the load-bearing capacity $R_{\perp d}\left[\mathrm{~N} / \mathrm{mm}^{2}\right]$ |
| :---: | :---: | :---: |
| Point and strip bearings | 0.83-2.33 | $\mathrm{R}_{\perp \mathrm{d}}=5.3805 \cdot \mathrm{~S}-0.6536$ |
|  | 2.33-2.50 | $\mathrm{R}_{\mathrm{Ld}^{\prime}}=10.635 \cdot \mathrm{~S}-12.89$ |
|  | 2.50-5.00 | $\mathrm{R}_{ \pm d}=8.4004 \cdot \mathrm{~S}-7.3293$ |
|  | $\geq 5.00$ | $\mathrm{R}_{\perp \mathrm{d}}=34.7$ |

Table 2:
$\mathbf{R}_{\perp_{d}}=$ Rated value of the associated load-bearing capacity of the bearing [ $\mathrm{N} / \mathrm{mm}^{2}$ ] perpendicular to the bearing plane as a function of the shape factor $S$ at a compression $\varepsilon=40 \%$.

## ESZ type 100 | with national technical approval General information and calculation basis

## Initial assumptions

| $\mathbf{F}_{\text {Z,max,d }}$ | $=185$ | kN |
| :--- | :--- | :--- |
| $\mathbf{f}_{\text {tRd }}$ | $=$ Formula | $\mathrm{N} / \mathrm{mm}^{2}$ |
| $\mathbf{a}$ | $=130$ | mm |
| $\mathbf{b}$ | $=150$ | mm |
| $\mathbf{t}$ | $=15$ | mm |
| $\mathbf{G}$ | $=1.2$ | $\mathrm{~N} / \mathrm{mm}^{2}$ |
| $\boldsymbol{\alpha}_{\text {Statics }}$ | $=5.2$ | $\%$ |
| $\boldsymbol{\alpha}_{\text {Obliqueness }}$ | $=10$ | $\%$ |
| $\boldsymbol{\alpha}_{\text {Unevenness }}$ | $=4.8$ | $\%$ |
| $\boldsymbol{\alpha}_{\text {bd total }}$ | $=20$ | $\%$ |
| $\boldsymbol{n}_{2}$ | $=0.215$ |  |

In this calculation example, a rotation around the axis parallel to the bearing side $b\left(\alpha_{b d}\right)$ is calculated. The bearing has no holes.

## Calculation wax

$$
\begin{aligned}
& \mathrm{S}=\frac{130 \cdot 150}{2 \cdot 15 \cdot(130+150)}=2.32 \\
& \mathrm{R}_{\perp \mathrm{d}}=5.3805 \cdot \mathrm{~S}-0.6536=17.17 \cdot 2.32-0.6536=11.83 \mathrm{~N} / \mathrm{mm}^{2} \\
& \mathrm{f}_{\mathrm{tRd}}=\frac{11.83}{0.215} \cdot \frac{15}{130}=6.35 \mathrm{~N} / \mathrm{mm}^{2} \\
& \sigma_{\mathrm{z}, \mathrm{Rd}}=\left[6.35-0.02 \cdot \frac{1.2}{2} \cdot\left(\frac{130}{15}\right)^{2}-0 \cdot \frac{1.2}{2} \cdot\left(\frac{150}{15}\right)^{2}\right] \cdot \frac{130}{15} \cdot 0.215=\mathbf{1 0 . 1 5} \mathrm{N} / \mathrm{mm}^{2} \\
& \sigma_{\mathrm{z}, \mathrm{~m}}=\frac{185.000}{130 \cdot 150}=\mathbf{9 . 4 9 \mathbf { N } / \mathrm { mm } ^ { 2 }} \\
& \sigma_{\mathrm{z}, \mathrm{Rd}}=\mathbf{1 0 . 1 4 ~ N} / \mathrm{mm}^{2} \geq \sigma_{\mathrm{m}}=9.49 \mathrm{~N} / \mathrm{mm}^{2}>\text { Proof provided! }
\end{aligned}
$$

